

# Asymptotic and Numerical Techniques for Evaluating the Chemical and Thermal Influences of Fire-Suppressants on Nonpremixed Flames

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## Abstract

Analytical and numerical approaches for evaluating the chemical and thermal influence of fire suppressants on laminar nonpremixed flames are described here. The analytical approaches employ asymptotic techniques for evaluating the chemical and thermal influences of suppressants on flames. In asymptotic analysis the Damkohler number  $\delta \equiv \tau_f / \tau_c$  is used to characterize the structure of the flame. Here  $\tau_f$  and  $\tau_c$  are respectively the characteristic flow time and the characteristic chemical reaction time. The results of asymptotic analysis show flame extinction to take place at a critical value of the Damkohler number represented by  $\delta_c$ . Combustion takes place for values of  $\delta$  greater than  $\delta_c$ . Chemical influences of any agent increases the value of  $\delta_c$ . On the other hand thermal influences do not change the value of  $\delta_c$ . Using this criteria  $\text{CF}_3\text{Br}$  is found to have significant influence on flame chemistry and  $\text{N}_2$  is found to extinguish the flame by thermal mechanisms. The thermal and chemical influences of a number of replacement agents are evaluated using this criteria and the results are compared with those for  $\text{CF}_3\text{Br}$  and  $\text{N}_2$ .

A numerical and analytical technique is described for evaluating the effectiveness of a given agent in extinguishing nonpremixed flames when it is mixed with air with its effectiveness when it is mixed with fuel. This technique is applied to evaluate the effectiveness of  $\text{CF}_3\text{Br}$  and  $\text{N}_2$  in extinguishing methane flames. The results show  $\text{CF}_3\text{Br}$  to be more effective in extinguishing the flame when it is mixed with air in comparison with its effectiveness when it is mixed with fuel.

## 1 Introduction

Numerous agents are currently being tested to find out if they can replace Halon 1301 ( $\text{CF}_3\text{Br}$ ) for firefighting. These replacement agents extinguish fires by chemical and thermal mechanisms. It is of practical interest to evaluate the degree of chemical and thermal influences of these agents on fires. Here previously used experimental, numerical and analytical approaches for evaluating the levels of chemical and thermal influences of any given fire-suppressant on

flames are described [1-3].

The experimental studies involve measurements on nonpremixed flames stabilized in the counterflowing configuration. This configuration provides a convenient geometry for detailed, fundamental studies of the structure and mechanisms of extinction of nonpremixed flames. In the counterflowing configuration the reactant streams (fuel stream and the oxidizer stream) flow toward each other, and a stagnation plane is formed where the value of the normal component of the flow velocity is zero. For hydrocarbon fuels, the stoichiometry of the overall combustion process is such that the flame is stabilized on the oxidizer side of the stagnation plane. In the counterflowing configuration, the flame front (defined as the position where the temperature attains a maximum value) is approximately parallel to the stagnation plane, and the structure of the flame in the vicinity of the stagnation streamline is nearly *one-dimensional*. Also in the counterflowing configuration, the characteristic residence time for the reactants in the flame can be easily related to the flow velocity of the reactants. This simplifies theoretical interpretation of the experimental results. Fire suppressants that extinguish flames by predominantly chemical mechanisms have different effectiveness when added to the fuel stream or to the oxidizer stream of the nonpremixed flame. Here previously used experimental, numerical and analytical approaches for comparing the effectiveness of chemical fire suppressants when it is added to the oxidizer stream with its effectiveness when added to the fuel stream is also discussed [4,5].

## 2 Thermal and Chemical Influences of Flame Suppressants

Experimental data showing extinction characteristics of various suppressants can be used to distinguish between their thermal and chemical influences on the flame. Recent studies [2,3] have examined the chemical and thermal influences of various halogenated agents on the structure of nonpremixed flames. These agents were tested to find out if they are suitable substitutes for Halon 1301 ( $\text{CF}_3\text{Br}$ ). The agents considered were  $\text{CF}_3\text{CH}_2\text{CF}_3$  (HFC-236fa),  $\text{CHFClCF}_3$  (HCFC-124),  $\text{CHF}_2\text{CF}_3$  (HFC-125),  $\text{C}_3\text{HF}_7$  (HFC-227ea),  $\text{C}_3\text{F}_8$  (FC-218),  $\text{CHF}_2\text{Cl}$  (HCFC-22),  $\text{CH}_2\text{FCF}_3$  (HFC-134a),  $\text{C}_4\text{F}_{10}$  (FC-31-10),  $\text{CH}_2\text{F}_2/\text{CHF}_2\text{CF}_3$  (60%/40% by volume) (HFC-32/HFC-125), *cyclo*- $\text{C}_4\text{F}_8$  (FC-318) and  $\text{C}_2\text{F}_6$  (FC-116). The counterflowing configuration was employed and the fuels used were heptane and JP-8 [2,3]. The effectiveness of these agents in extinguishing nonpremixed flames burning these fuels were compared with those of  $\text{N}_2$  and  $\text{CF}_3\text{Br}$ . The oxidizing gas was a mixture of air and the agent. Experimental results were interpreted using one-step, activation-energy asymptotic theories [6-9]. Although this is a poor approximation for the chemistry taking place in the reaction zone, it provides a rough indication of the thermal and chemical influence of these agents on the flame structure.

### 2.1 Experimental Results

The counterflow burner employed in the present study is described in detail elsewhere [3]. It consists of a fuel cup which has a diameter of 45 mm and a depth of 18 mm, and an oxidizer duct with an inner diameter of 50.1 mm through which the oxidizing stream containing the gaseous agent to be tested is introduced into the flame. The oxidizer duct has approximately

five layers of fine wire screens to reduce turbulence and ensure a flat velocity profile at the exit of the duct. Experiments were performed with  $L = 1$  cm, where  $L$  is the separation distance between the surface of the liquid pool and the exit of the oxidizer duct. The flowrates of air, nitrogen and the gaseous agent were measured accurately by use of variable area flowmeters. The volumetric flow rates of the gases measured using these flowmeters are rated at an accuracy of 3%. Experiments were performed with the oxidizer stream at an initial temperature of 298K. To obtain the critical conditions of extinction the flow rate of air in the oxidizer duct was maintained at some predetermined value and the liquid pool was ignited. The agent was then added gradually to the oxidizer stream until the flame extinguished. The flowrates of air and the agent at extinction were recorded. The experiments were repeated for a different value of the flowrate of air. The velocity of the oxidizer stream at the exit of the oxidizer duct  $U$  was presumed to be equal to the ratio of the total flow rate to the cross sectional area of the duct. The characteristic strain rate at extinction was calculated using the expression  $a = 2U/L$  [10].

Figure 1 shows the mass fraction of the agents at extinction as a function of the strain rate.

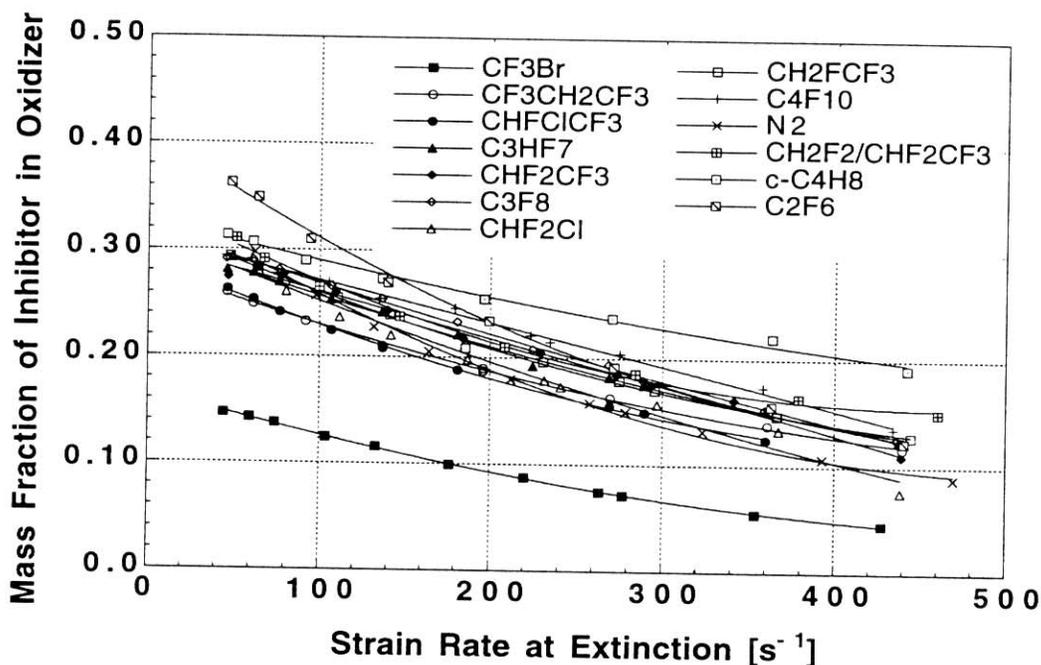


Figure 1: Mass fraction of various agents as a function of the strain rate ( $s^{-1}$ ) at extinction. The fuel tested was heptane and the initial temperature of the oxidizer stream was  $25^{\circ}C$  [2].

For a given agent, the region above the curve represents flammable mixtures. At values of  $a$  below those shown in Fig. 1, it was not possible to stabilize a steady flame in the counterflow burner. Figure 1 shows that for any given value of  $a$ , the value of the mass fraction of  $CF_3Br$  required to extinguish the flame is lower than the mass fraction of all other agents. Therefore on a mass basis  $CF_3Br$  is considerably more effective in extinguishing the flame than all the agents tested. Figure 1 shows that the relative effectiveness of various agents in extinguishing the flame changes for different values of  $a$ . At low strain rates  $CF_3CH_2CF_3$  is most effective

and  $C_2F_6$  is least effective in extinguishing the flame, whereas at high strain rates  $CHF_2Cl$  is most effective and cyclo- $C_4F_8$  is least effective in extinguishing the flame. In fact at high strain rates only  $CF_3Br$  is more effective than  $N_2$  in extinguishing the flame and the effectiveness of  $CHF_2Cl$  is nearly the same as that of  $N_2$ . Figure 1 shows that, except for  $CF_3Br$ , the mass-based effectiveness of the various agents in extinguishing the flame does not differ much from that of nitrogen. If the results in Fig. 1 are plotted showing the mole fractions of the agents at extinction as a function of  $a$ , then the relative effectiveness of the agents ranked on a mole basis is found to be different from their mass-based effectiveness. On a mole basis  $CF_3Br$  is found to be most effective and  $N_2$  least effective [3].

## 2.2 Interpretation of the Experimental Results Using Activation-Energy Asymptotic Theories

Activation-energy asymptotic theories of flame extinction [6-9], where the gas-phase chemical reaction is approximated as a one-step process have predicted the value of the Damkohler number at extinction  $\delta_e$ , which is defined as the ratio of  $\tau_f$  to  $\tau_c$ , where  $\tau_f$  is the characteristic flow time and  $\tau_c$  the characteristic chemical reaction time. For a given chemical system, the value of the  $\tau_c$  will depend on the relative concentrations of the various reactants including that of the agent and the local gas temperature, and the value of  $\tau_f$  will depend on the strain rate  $a$ . For a given fuel-oxidizer system, the effectiveness of an agent in interfering in the chemical reaction between the fuel and oxygen can be roughly estimated by examining the value of  $\delta_e$ . For a given value of  $\tau_f$ , the value of  $\tau_c$  at extinction can be expected to decrease and the corresponding value of  $\delta_e$  to increase with increasing chemical inhibition of the agent. In absence of temperature measurements in the flame at conditions close to flame extinction, the calculated adiabatic flame temperature  $T_f$ , can be expected to provide a rough measure of the characteristic chemical time. The value of  $\tau_c$  decreases with increasing values of  $T_f$ .

Measurements of critical conditions of heptane flames with some selected agents shown in Fig. 1 were used to calculate the adiabatic flame temperature and the results are plotted in Fig. 2. In the calculations, the agents were presumed to be inert, and the gas-phase chemical reaction was presumed to occur as a one-step overall process with stoichiometric amounts of fuel reacting with oxygen to form the products  $CO_2$  and  $H_2O$ . Figure 2 shows that for any given value of the strain rate  $a$ , the value of  $T_f$  calculated for flames inhibited with  $CF_3Br$  is higher than the value of  $T_f$  calculated for flames inhibited with the other agents. Therefore for any value of  $\tau_f$ , the critical conditions of extinction measured with  $CF_3Br$  yield values of  $\tau_c$  which is smaller, than those for the other agents. Hence the values of  $\delta_e$  for extinction of flames with  $CF_3Br$  is higher than those for extinction of flames with the other agents. Therefore  $CF_3Br$  appears to interfere more with the flame chemistry than the other agents and cannot be presumed to be inert. Also, except  $CF_3Br$ , the values of  $T_f$  calculated using the measured mass fractions of the other agents at extinction do not differ much from those calculated using the measured mass fraction of  $N_2$  at extinction. Since the influence of  $N_2$  on the flame is primarily thermal, it appears that with the exception of  $CF_3Br$ , all the agents have a significant thermal influence on the flame.

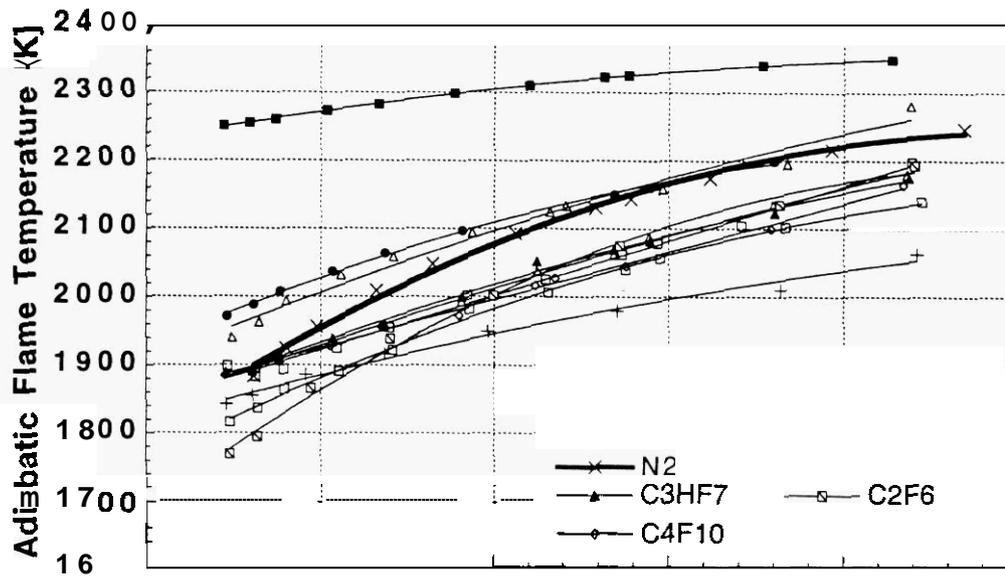


Figure 2: Calculated adiabatic flame temperature as a function of the strain rate ( $s^{-1}$ ) at extinction [2].

The calculated values of  $T_f$  are used to interpret the extinction measurements using predictions of activation-energy asymptotic analysis [6-9]. The agents, including  $CF_3Br$ , presumed to be chemically inert. The rate of gas-phase chemical reaction between the fuel and oxygen is represented by the Arrhenius expression  $k = B \exp[-E_0/(RT)]$  where  $k$  is the rate constant of the reaction,  $B$  the frequency factor,  $E_0$  the activation energy,  $R$  the gas constant and  $T$  the gas temperature. The activation energy of the chemical reaction is presumed to be large in comparison to the thermal energy in the flame. Results of activation-energy asymptotic analysis predict that at flame extinction [8]

$$Fa/T_f^2 = \rho_\infty T_\infty B_0 R^3 \exp[-E_0/(RT_f)] / (2W_F E_0^3) \quad (1)$$

where  $\rho_\infty$  and  $T_\infty$  represent respectively the density and temperature of the oxidizer stream at the exit of the gas duct,  $W_F$  is the molecular weight of the fuel, and  $B_0$  represents the frequency factor for molar rate of consumption of oxygen. The value of the quantity  $F$  depends on the flow field, and Krishnamurthy et al [8] have outlined methods for estimating  $F$  for the counterflow geometry. It follows that a plot of  $\ln(Fa/T_f^2)$  versus  $1/T_f$  represents an Arrhenius type plot, and the slope of this plot can be used to deduce the activation energy of the chemical reaction between the fuel and oxygen.

Figure 3 shows Arrhenius diagrams plotted using the measurements of critical conditions of extinction of heptane flames with some selected agents shown in Fig. 1. The value of  $E_0$

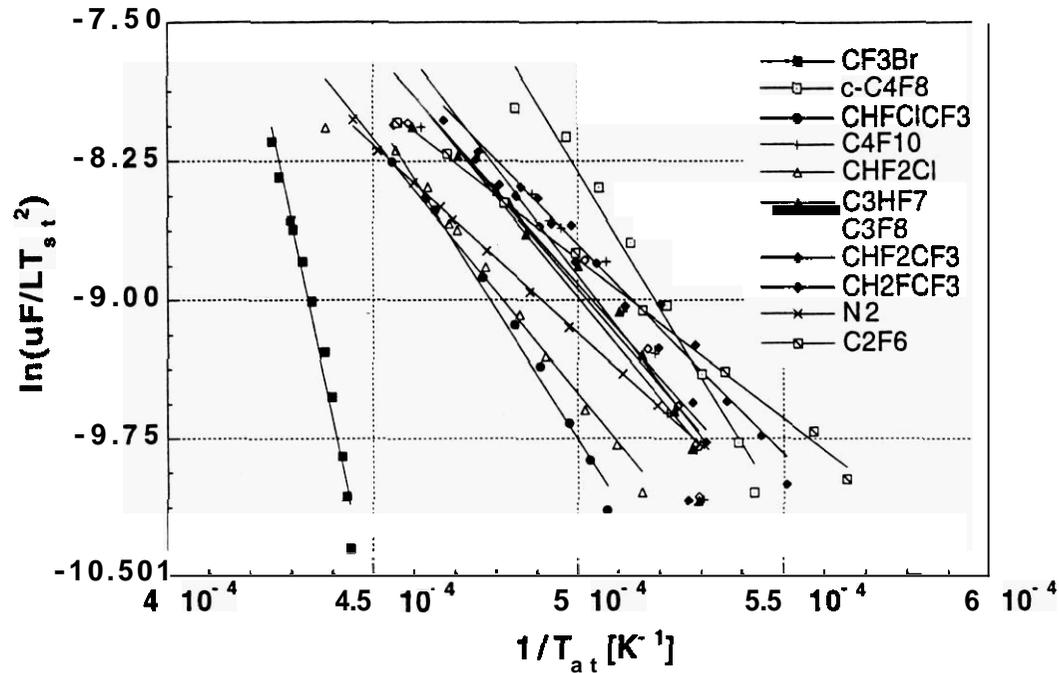


Figure 3: Arrhenius plots obtained using the extinction data shown in Fig. 1 [2] .

deduced from the slopes of the lines in Fig. 3 is shown in Table 1.

The overall activation energy for one step reaction between heptane vapors and oxygen is 168 kJ/mole. Since the influence of N<sub>2</sub> on the flame is primarily thermal deviations of the value of  $E_0$  calculated using the extinction data obtained with other agents from that of N<sub>2</sub> is a rough measure of the chemical influence of these agents. Since the value of  $E_0$  calculated using extinction data with CF<sub>3</sub>Br is unrealistically high, clearly the one-step approximation is not valid, and CF<sub>3</sub>Br clearly appears to have a significant influence on the flame chemistry. However, in comparison to CF<sub>3</sub>Br, the other agents appear to have considerably less influence on the flame chemistry.

### 3 Comparison of the Effectiveness of CF<sub>3</sub>Br When Added to the Oxidizer Stream With Its Effectiveness When Added to the Fuel Stream

A detailed experimental, numerical and analytical study was conducted previously to clarify the influence of CF<sub>3</sub>Br on the structure and critical conditions of extinction of nonpremixed methane flames [4,5,11,12]. In some of these previous studies the effectiveness of CF<sub>3</sub>Br in extinguishing methane flames when it is added to the fuel stream was compared with its effectiveness when it is added to the oxidizer stream [4,5]. The counterflowing configuration was used. The burner used in the experiments is described elsewhere [5]. It comprised two opposing ducts with inner diameters of 22.2 mm through which the gaseous reactants were introduced. The distance  $L$  between the exits of the ducts was 10 mm. Seshadri and Williams [10] have shown that for

Agent	Chemical Formula	$E_0$ in kJ/mole
Halon 1301	$CF_3Br$	899
FC-318	cyclo- $C_4F_8$	311
HCFC-124	$CHFClCF_3$	294
FC-31-10	$C_4F_{10}$	244
HCFC-22	$CHF_2Cl$	231
HFC-227ea	$C_3HF_7$	237
FC-218	$C_3F_8$	218
HFC-125	$CHF_2CF_3$	214
HFC-134a	$CH_2FCF_2$	189
Nitrogen	$N_2$	168
FC-116	$C_2F_6$	143

laminar flow in the counterflowing configuration, if the Reynolds number is large enough, and if the values of the tangential component of the flow velocities at the exits of ducts are zero, then the flow field comprises two inviscid rotational regions on either side of a thin viscous region formed in the vicinity of the stagnation plane. The strain rate  $a$ , defined as the normal gradient of the normal component of the flow velocity in the inviscid region on the oxidizer side of the stagnation plane, is given by the expression [10]

$$a = \frac{2|V_2|}{L} \left( 1 + \frac{|V_1|\sqrt{\rho_1}}{|V_2|\sqrt{\rho_2}} \right), \quad (2)$$

where  $V$  is the velocity, and  $\rho$  the density. Subscripts 1 and 2 respectively denote conditions in the fuel stream and in the oxidizer stream at the exit plane of the ducts. The value of the strain rate calculated from Eq. 2 was taken as a measure of the inverse of the characteristic flow time of the reactants in the flame. The value of the strain rate at extinction is denoted by  $a_q$ .

Numerical calculations were performed to determine the critical conditions of extinction and the results were compared with measurements [4, 5]. In the numerical calculations the oxidation of methane was described using reactions 1-35, 37-40 shown in Table 1 of Ref. [13]. This mechanism excludes reactions in which compounds comprising two or more carbon atoms participate. Westbrook [14] has proposed a detailed chemical-kinetic mechanism for describing the influence of  $CF_3Br$  on the structure of premixed flames of hydrogen, methane, methanol, and ethene. The inhibition chemistry was described by reactions B1-B10, B15, B17, B18, B21-B36, B40-B46, B48-B56 taken from Table II of Ref. [14]. For a given value of the strain rate two converged solutions can be obtained. The solution with the higher value of the maximum flame temperature is stable and the other is unstable. The calculations thus yield the classical "C-shaped" curve when the maximum value of the flame temperature is plotted as a function of the strain rate. For values of the strain rate greater than a critical value  $a$ , no solution is

obtained. **This** critical value  $a$ , is the strain rate at extinction.

In Fig. 4 the strain rate  $a_q$  is plotted as a function of the mass fraction of the inhibitor,  $Y_{I,2}$ , in the oxidizer stream, where the subscript Z denotes  $CF_3Br$ . The points in Fig. 4 represent

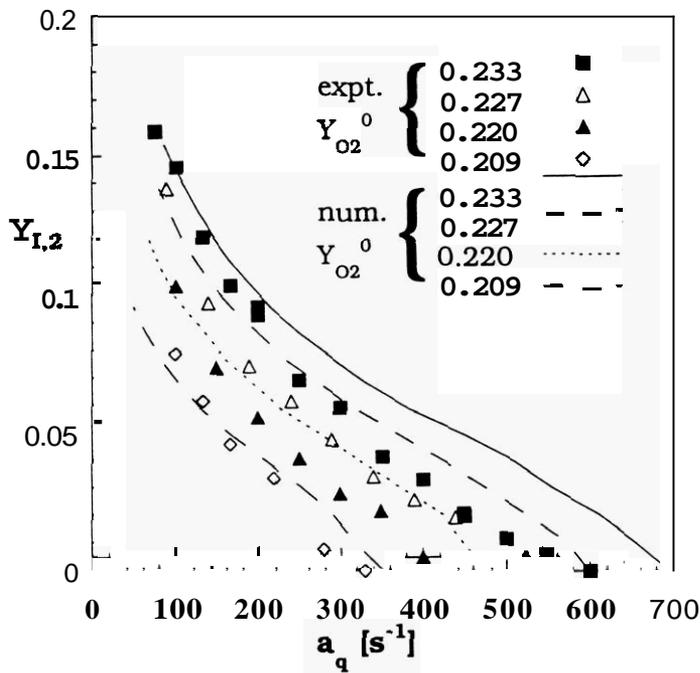


Figure 4: Comparison of experimental measurements (points) of critical conditions of extinction with results of numerical calculation (lines) with the inhibitor added to the oxidizer stream. Measurements are made for values of  $Y_{O_{2,2}}^0/Y_{F,1}^0$  equal to 0.233. The values of  $Y_{O_{2,2}}^0$  prior to addition of the inhibitor is shown [5] .

measurements and the lines are results of numerical Calculation with detailed chemistry. In this figure the region below any curve represent flammable mixtures. The fuel stream and the oxidizer stream were diluted with nitrogen. The measurements were made with the value of the ratio  $Y_{O_{2,2}}^0/Y_{F,1}^0$  fixed at 0.233 and for values of  $Y_{O_{2,2}}^0$  equal to 0.233, 0.227, 0.220, and 0.209. Here  $Y_{O_{2,2}}^0$  is the mass fraction of oxygen at the oxidizer boundary prior to addition of the inhibitor, and  $Y_{F,1}^0$  is the mass fraction of fuel at the fuel boundary prior to addition of the inhibitor. The numerical calculations were performed with the composition of the reactants streams identical to those used in the experiments and for various values of the strain rates [5]. Figure 4 shows the results of numerical calculations to agree very well with the measurements. In general at a given value of  $Y_{I,2}$  the calculated value of  $a$ , is larger than its measured value. However for high inhibitor concentrations (low strain rates) at a given value of  $Y_{I,2}$  the calculated value of  $a$ , is very close to its measured value.

Figure 5 shows  $a_q$  as a function of the mass fraction of  $CF_3Br$  in the fuel stream,  $Y_{I,1}$  [5]. The points in Fig. 5 represent measurements and the lines are results of numerical calculation with detailed chemistry. The measurements are made with the value of  $Y_{O_{2,2}}^0/Y_{F,1}^0$  fixed at

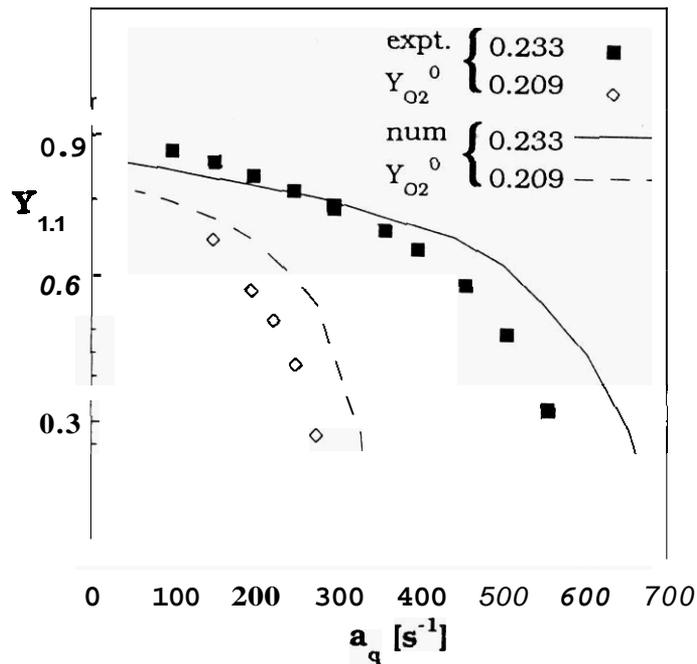


Figure 5: Comparison of experimental measurements (points) of critical conditions of extinction with results of numerical calculation (lines) with the inhibitor added to the fuel stream. Measurements are made for values of  $Y_{O_2,2}^0/Y_{F,1}^0$  fixed at 0.233. The values of  $Y_{O_2,2}^0$  prior to addition of the inhibitor is shown [5].

0.233 and for values of  $Y_{O_2,2}^0$  equal to 0.233 and 0.209. Figure 5 shows the results of numerical calculations to agree very well with the measurements.

A comparison of the results in Figures 4 and 5 show that for a given value of  $a$ , the measured and calculated values of  $Y_{I,1}$  are significantly larger than those of  $Y_{I,2}$ . To compare the effectiveness of  $CF_3Br$  in extinguishing the flame when it is added to the oxidizer stream with its effectiveness when added to the fuel stream, the flux of the inhibitor entering the reaction zone must be calculated. At the flamesheet which is presumed to be located at  $\xi_{st}$ , by definition, the mass fractions of fuel and oxygen calculated assuming frozen flow, must be in stoichiometric proportions [9,15]. Here  $\xi$  is a conserved scalar quantity and its value is 1 at the fuel boundary and 0 at the oxidizer boundary [9,15]. In a previous asymptotic study of chemical inhibition of nonpremixed methane flames by  $CF_3Br$  [11], it was presumed that in the flamesheet approximation [9],  $CH_4$  and  $CF_3Br$  are consumed in two different thin zones. The global chemistry at these thin zones were represented by [11]  $CH_4 + 2O_2 \rightarrow CO_2 + 2H_2O$ , and  $CF_3Br + 2H_2O \rightarrow HBr + 3HF + CO_2$ . It is noteworthy that oxygen does not appear in the global step that consumes  $CF_3Br$ . To the leading order it was shown that the position of the primary reaction zone  $\xi_{st}$  where fuel and oxygen are completely consumed is given by

$$\xi_{st} = [1 + 2Y_{F,1}W_{O_2}/(Y_{O_2,2}W_F)]^{-1}, \quad (3)$$

where  $Y_i$  and  $W_i$  represent the mass fraction and molecular weight of species  $i$  and the subscripts  $F$  and  $O_2$  denote the fuel and oxygen respectively. The mass fraction of the inhibitor does not appear in the expression for  $\xi_{st}$  because neither the fuel nor oxygen appears in the global step that consumes  $CF_3Br$ .

Following a previous study [16] the mass fraction of the inhibitor at the flamesheet,  $Y_{I,st}$  can be calculated from the expression  $Y_{I,st} = Y_{I,2} + (Y_{I,1} - Y_{I,2})\xi_{st}$ , assuming frozen flow and neglecting preferential diffusion, can be used to measure the effectiveness of the inhibitor. The values of  $Y_{I,st}$  calculated using the experimental results for  $Y_{O_2,2}^0 = 0.233$  in Figs. 4 and 5 are shown in Fig. 6. For a given value of strain rate the value of  $Y_{I,st}$  when the inhibitor is added

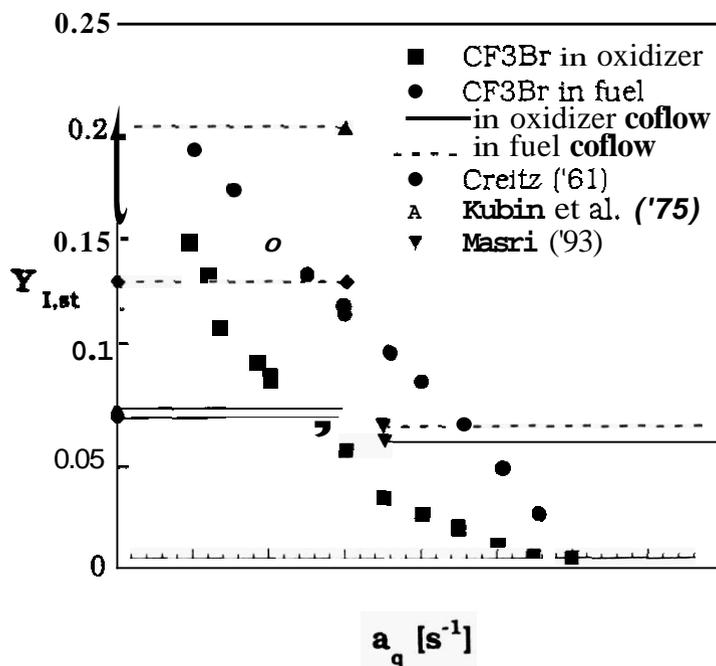


Figure 6: The strain rate at extinction  $a_q$  as a function of  $Y_{I,st}$  calculated using the extinction results for  $Y_{O_2,2}^0 = 0.233$ .

to the oxidizer stream is significantly lower than the value of  $Y_{I,st}$  when the inhibitor is added to the fuel stream. Therefore the experimental results show  $CF_3Br$  to be more effective in extinguishing the flame when it is added to the oxidizer stream. In Fig. 6 the values of  $Y_{I,st}$  calculated from results of previous experimental studies on laminar [17,18] and turbulent [16] flames stabilized in the coflowing configuration are also shown. In the coflow configuration the strain rate cannot be easily calculated from the velocities of the reactants, therefore the experimental results obtained from these previous studies are represented by straight lines in Fig. 6. The experiments of Creitz [17] and Kubin et al [18] were conducted on methane flames and the results show  $CF_3Br$  to be more effective in extinguishing the flames when it is added to the oxidizer stream. The experiments of Masri [16] were conducted on natural gas flames containing 90%  $CH_4$  by volume and show that the effectiveness of  $CF_3Br$  when it is added to

the oxidizer stream is nearly equal to its effectiveness when added to the fuel stream. The qualitative differences between the results obtained by Masri [16] and the present experimental results and those of Crietz [17] and Kubin et al [18] are attributed to differences in the flow field (turbulent flow versus laminar flow) and to the type of fuel (natural gas versus methane). These results clearly show  $\text{CF}_3\text{Br}$  to be more effective in extinguishing methane flames when it is added to the oxidizer stream

## 4 Summary

Experimental and analytical studies on mechanisms of extinction of heptane flames with a number of agents show that

1. On a mass and mole basis  $\text{CF}_3\text{Br}$  is considerable more effective in extinguishing the flames than all the substitute agents considered. With the exception of  $\text{CF}_3\text{Br}$ , on a mass basis the effectiveness of all the other agents in extinguishing nonpremixed flames was nearly the same as that of nitrogen.
2. Interpretation of the results based on one-step activation energy asymptotic theories shows that  $\text{CF}_3\text{Br}$  has a significant influence on the flame chemistry. The other agents tested appear to have a significant thermal and some chemical influence on the flame.

Experimental and numerical studies on mechanisms of extinction of methane flames with  $\text{CF}_3\text{Br}$  show that

1. The chemical kinetic mechanisms proposed by Westbrook [14] is found to provide a realistic description of inhibition of nonpremixed methane flames by  $\text{CF}_3\text{Br}$ .
2.  $\text{CF}_3\text{Br}$  is more effective in extinguishing nonpremixed methane flames when it is added to the oxidizer stream in comparison to its effectiveness when added to the fuel stream. This result is in agreement with previous studies [17,18].

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