

Measuring flow speed in a long-wavelength acoustic flowmeter



Keith A. Gillis

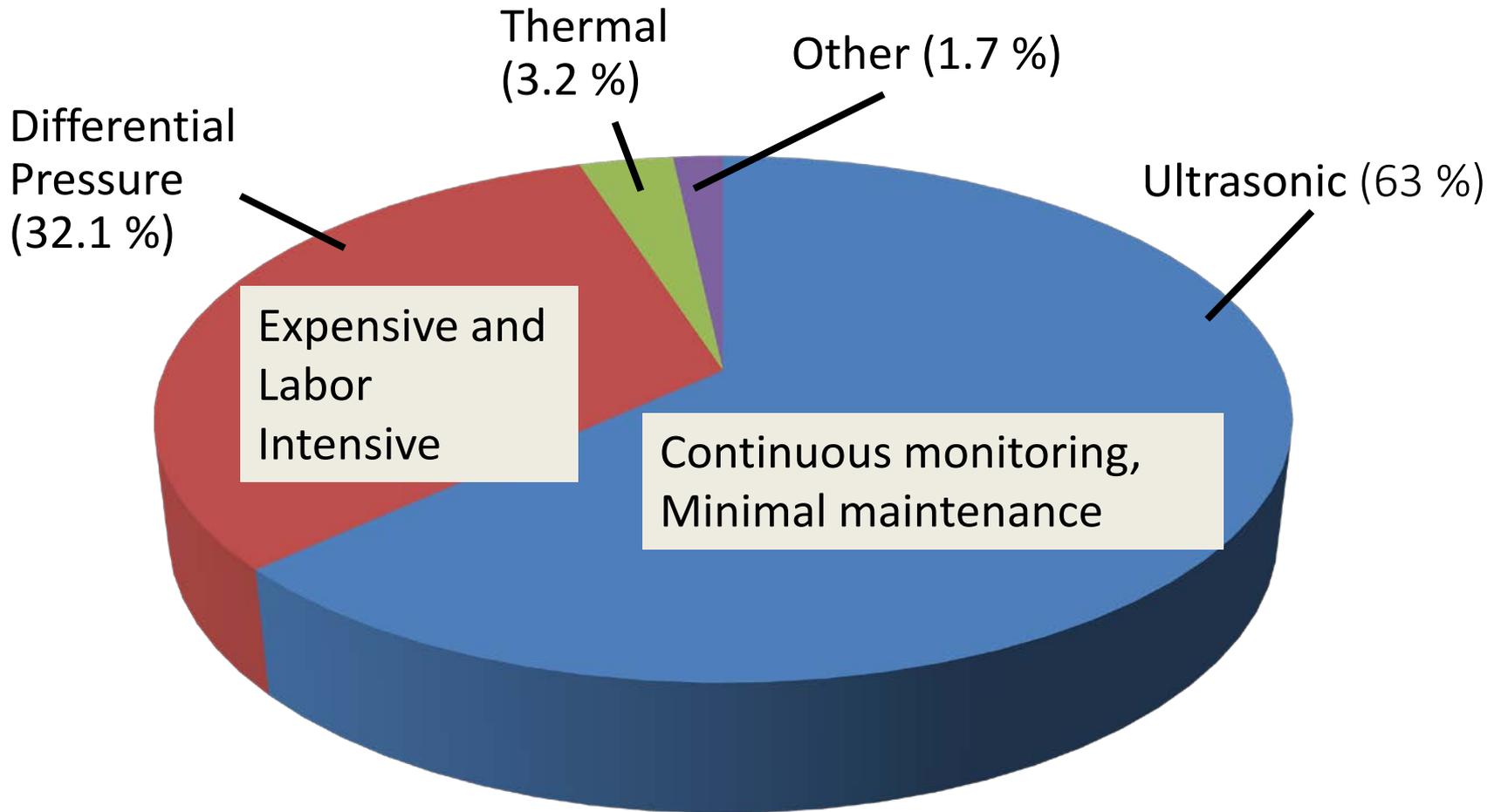
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Sensor Science Division
National Institute of Standards and Technology
Gaithersburg, MD*

Collaborators:

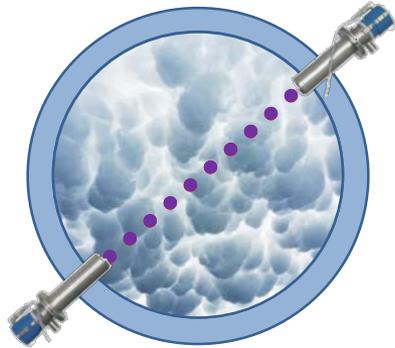
JohnPaul Abbott
Lee Gorny
Aaron Johnson
John Wright
Michael Moldover

NIST Greenhouse Gas Conference and Workshop
Gaithersburg, MD
June 28-29, 2017

Technologies for monitoring flow



Ultrasonic meters sample a small portion of the flow field



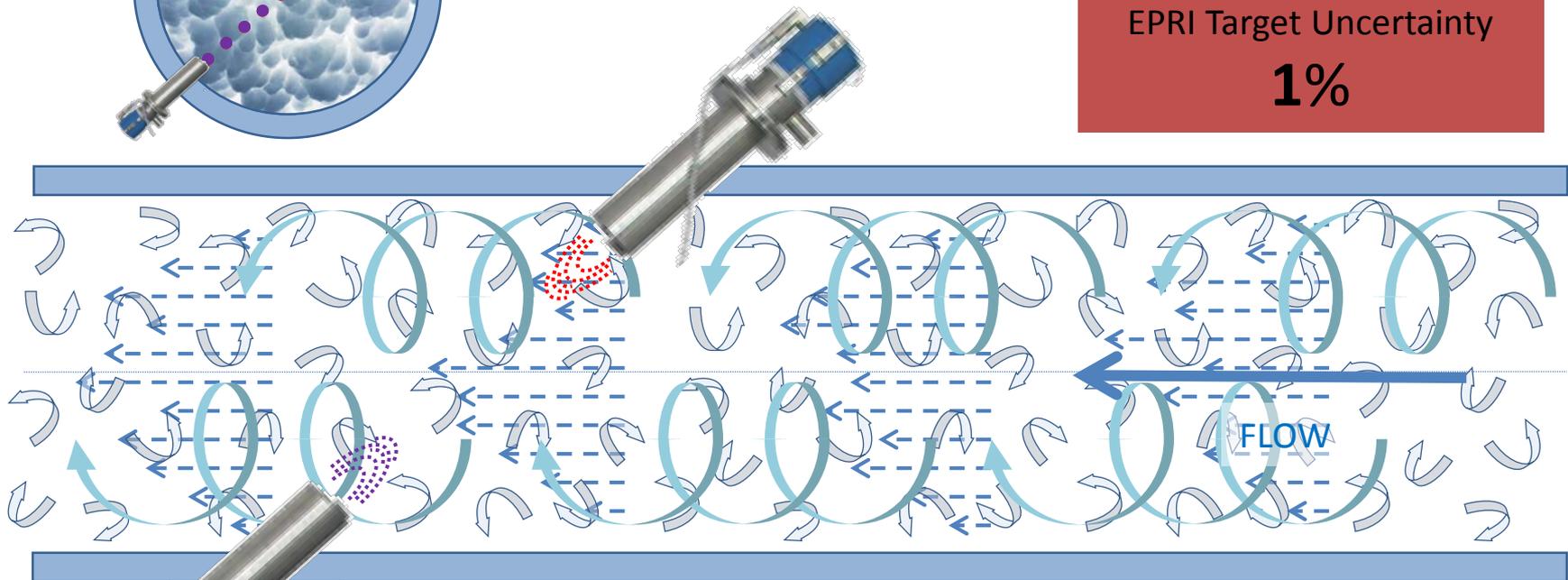
Estimated Uncertainty

5-20%

EPRI Target Uncertainty

1%

Open
End

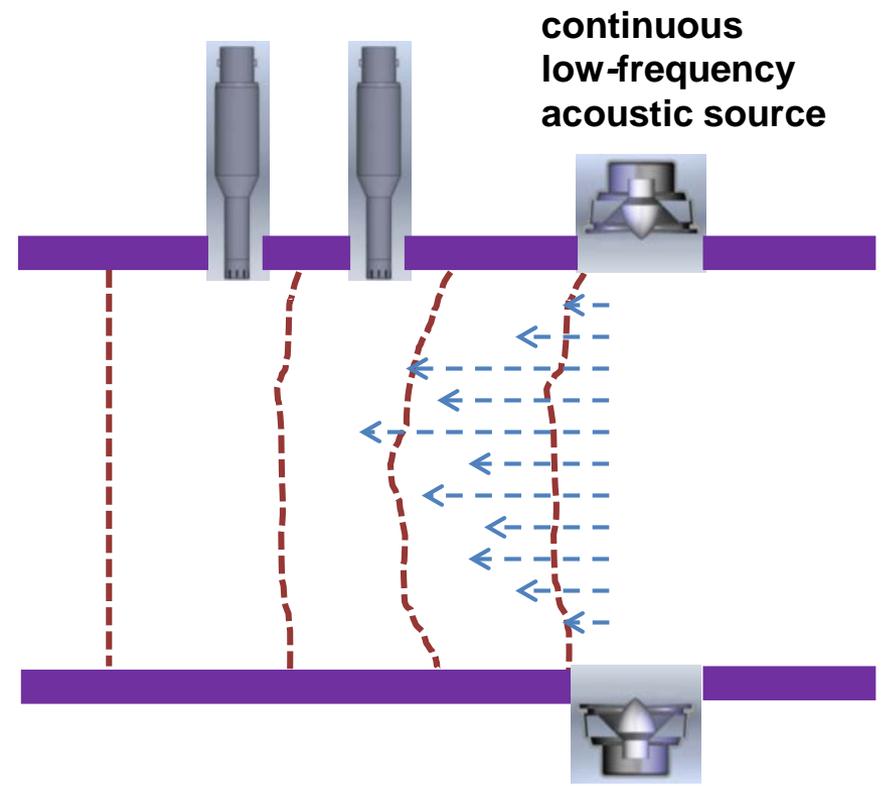
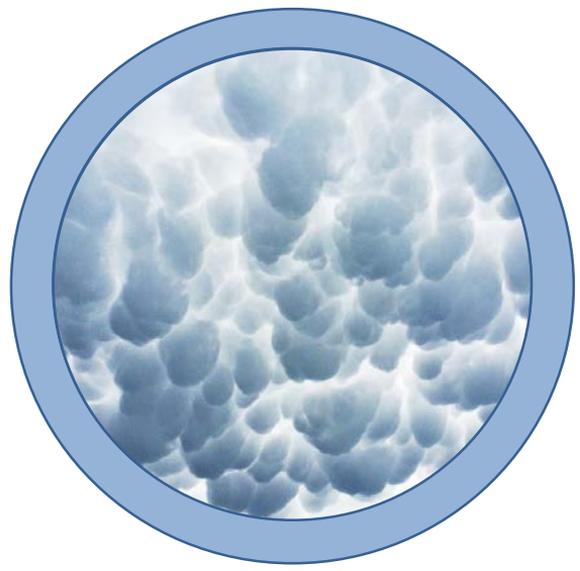


Multiple paths required

A long wavelength acoustic flowmeter (LWAF) measures flow with low frequency sound.

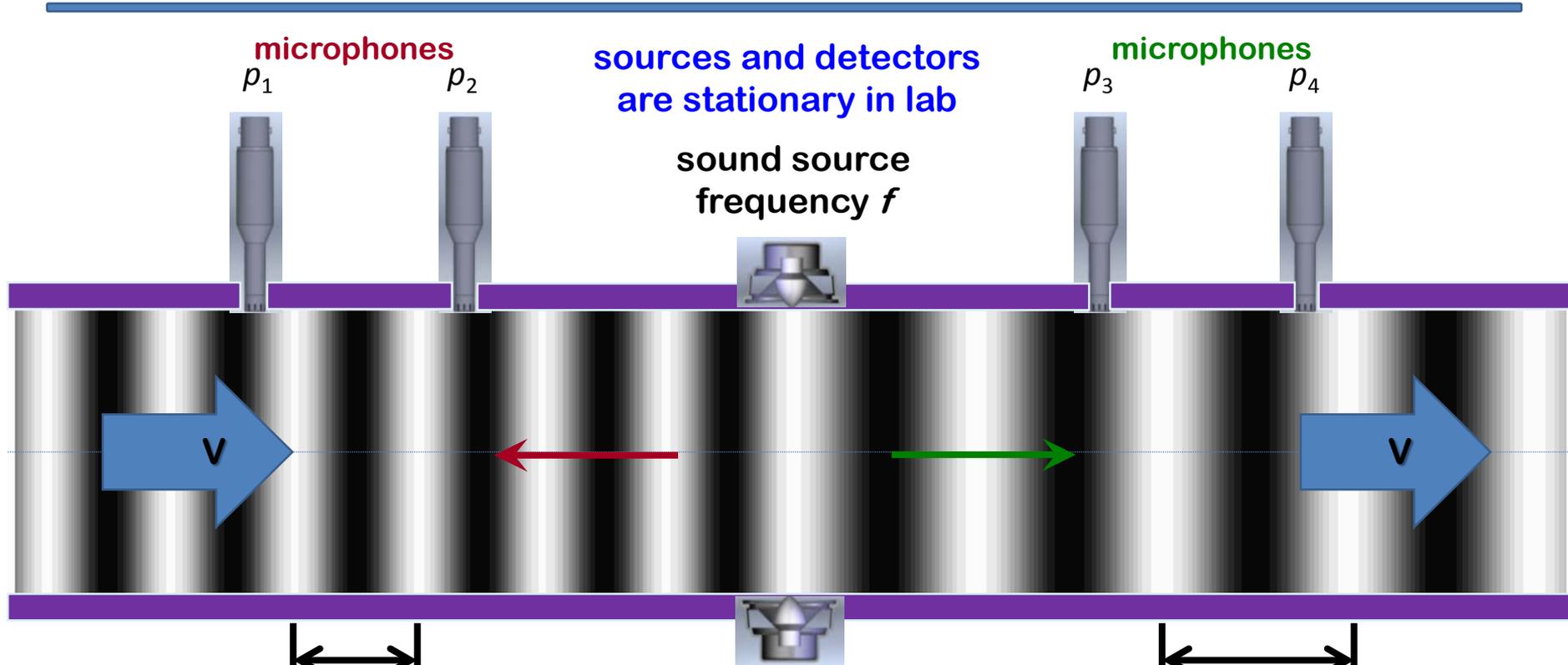
Plane wave propagation if $\lambda > 1.7 \times \text{diameter}$

Plane wave averages over cross-section



plane wave propagation is not affected by complex flow

Sound propagation in flow



$$\lambda_{\text{left}} = (c - V)/f$$

$$\text{Mach number } M = V/c$$

λ = wavelength

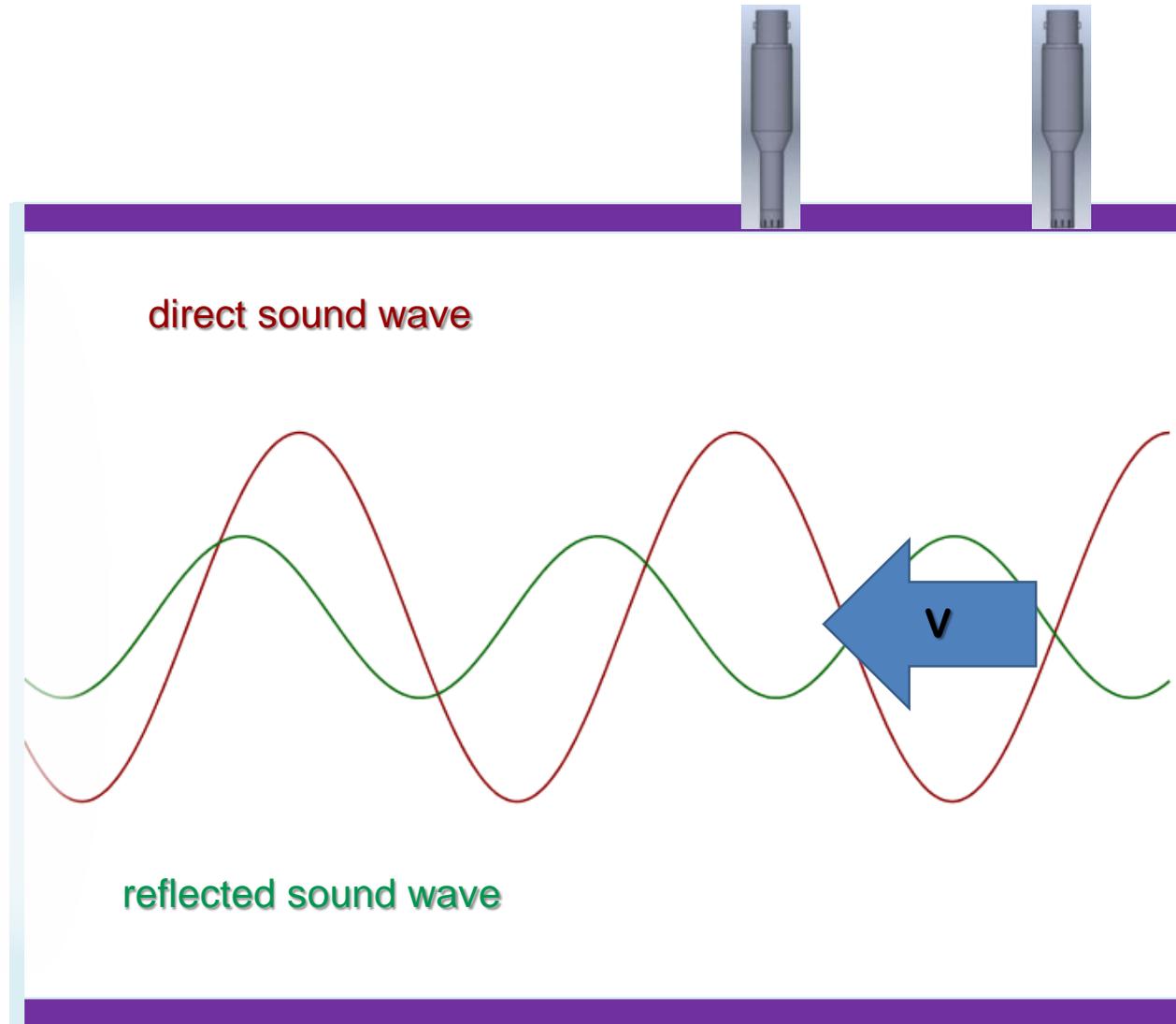
$$\lambda_{\text{right}} = (c + V)/f$$

$$\lambda_{\text{left}} = (1 - M) \lambda_0 < \lambda_{\text{right}} = (1 + M) \lambda_0$$

$$c = \frac{1}{2} (\lambda_{\text{right}} + \lambda_{\text{left}}) f \quad \text{and} \quad V = \frac{1}{2} (\lambda_{\text{right}} - \lambda_{\text{left}}) f$$

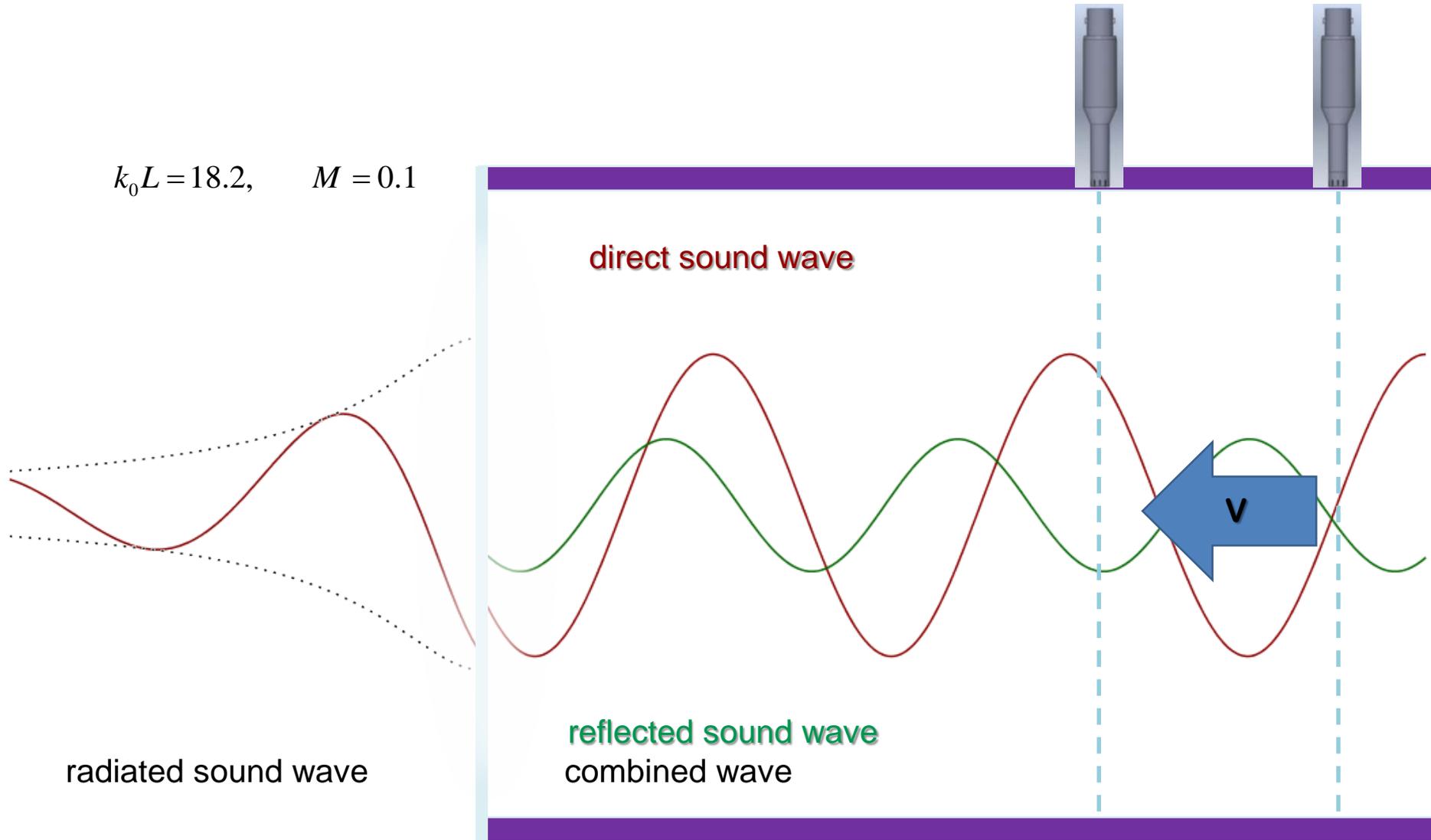
Partial standing waves in a duct with flow

$$k_0 L = 18.2, \quad M = 0.1$$

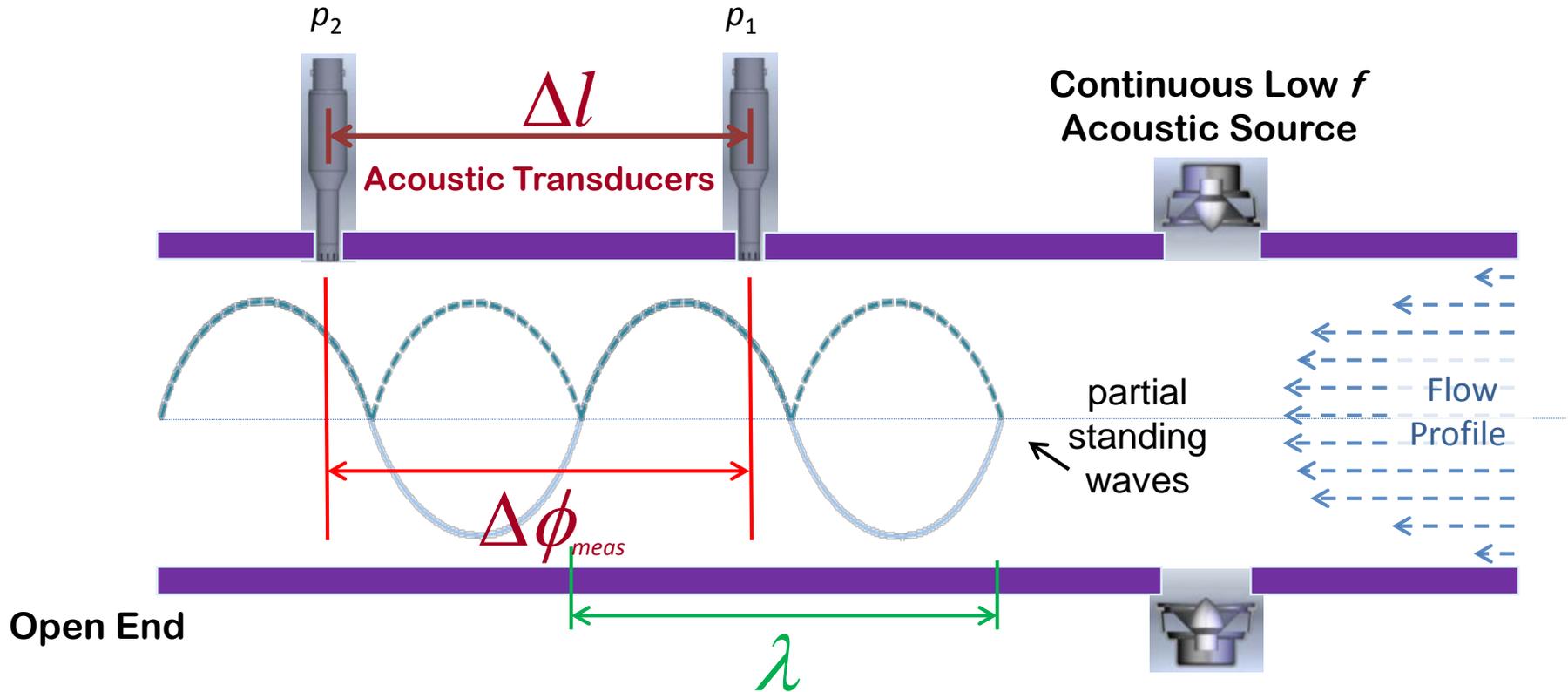


Partial standing waves in a duct with flow

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Acoustic measurements of flow



$$p_2/p_1 = |p_2/p_1| e^{i\Delta\phi}$$

λ - wavelength is proportional to the speed of sound.

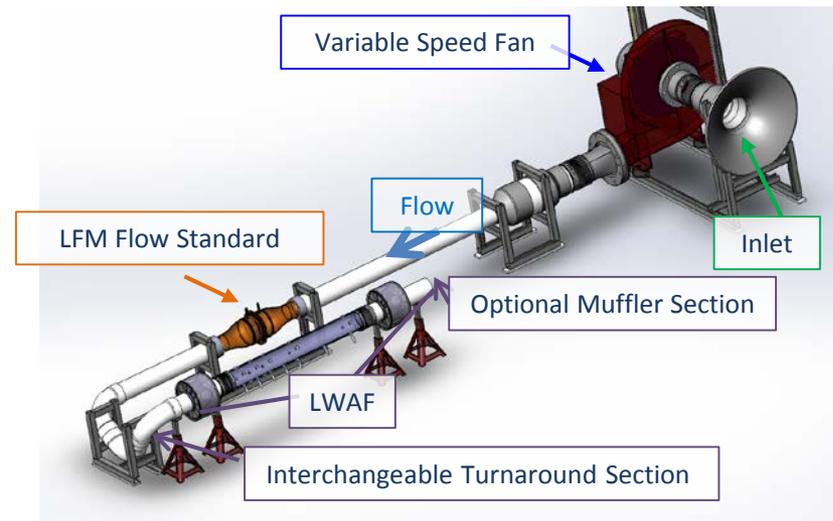
$$c_0 = 2f_n \Delta l / n$$

$\Delta\phi_{meas}$ - phase difference changes inversely with flow.

$$c_0 + V = 2\pi f_n \Delta l / \Delta\phi_n$$

LWAF with active sound generation: performance summary

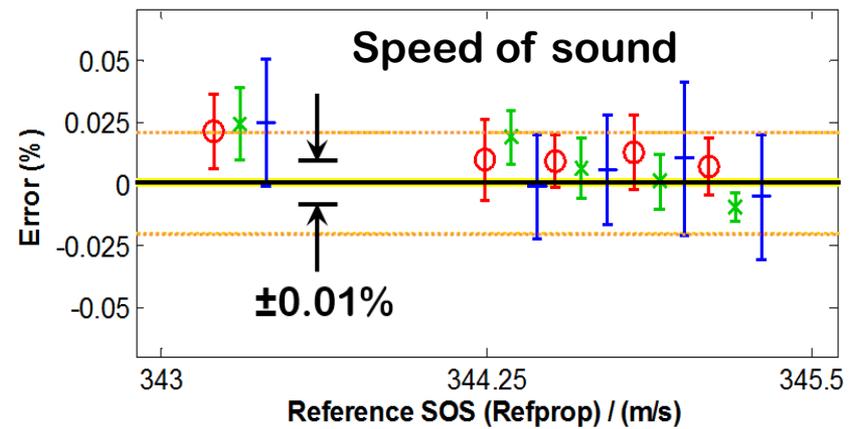
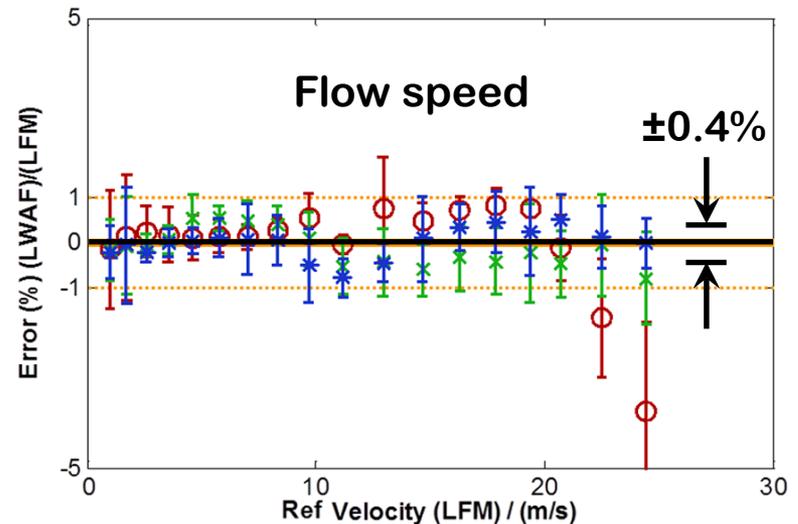
Previously, we assessed the performance and scalability of the NIST 1/100th scale (10 cm diameter) LWAF facility using an active sound source to generate long wavelength sound (frequency < 2 kHz)



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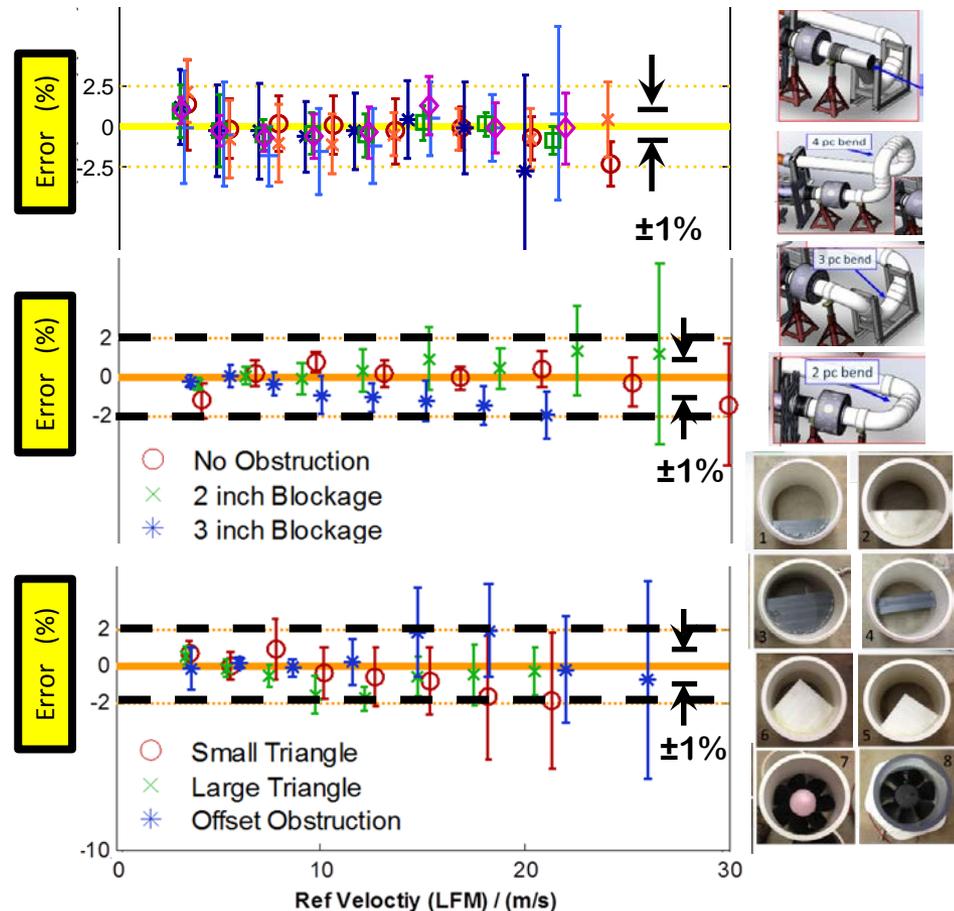
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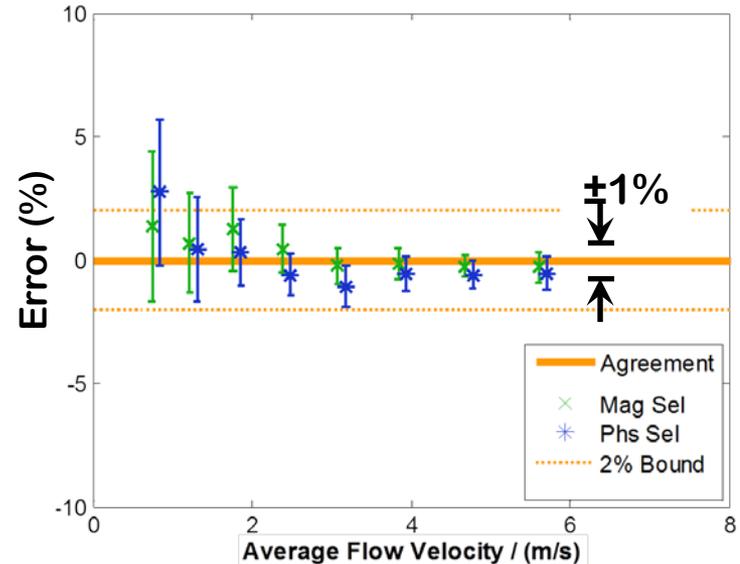
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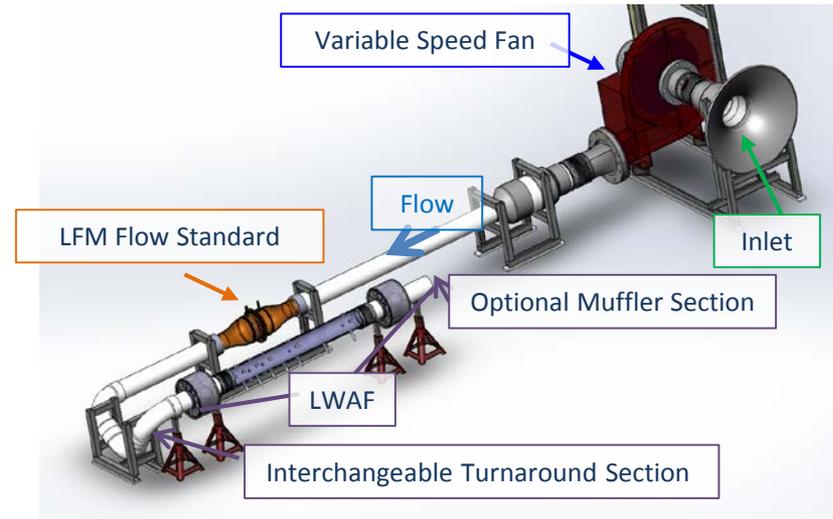
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It works!!!...

But... generating sound in a smokestack is impractical...

Sound generation in a smokestack

The volume of a typical smokestack is 10^6 x larger than the NIST 1/100th scale LWAF

- to achieve the same signal-to-noise ratio we need to generate 10^6 x higher sound pressure
- $20 \log(10^6) = 120$ dB higher SPL
- smokestacks are noisy environments
- required sound level $\sim 160 - 180$ dB SPL! (that's loud)



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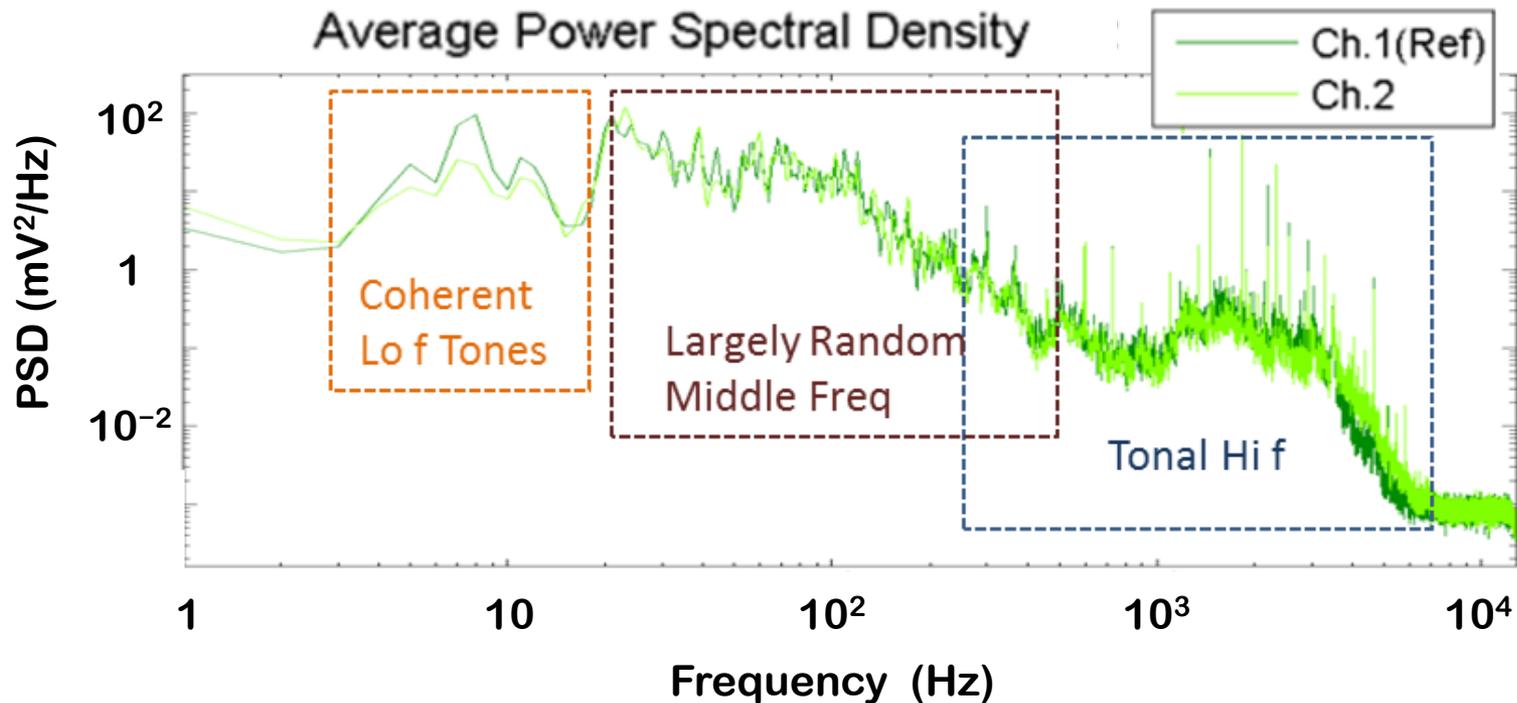
What is an alternative?

Acoustical noise in a power plant smokestack

Measurements at Mirant (GenOn) plant in Dickerson, MD

Smokestack diameter ≈ 10 m

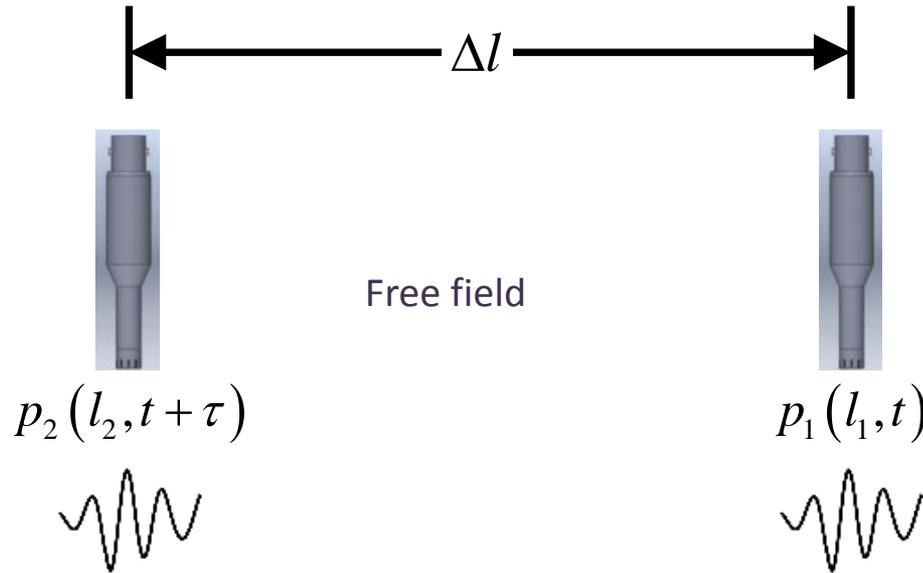
Plane wave sound for frequency < 20 Hz



Noise caused by blowers or turbulent flow.

Passive approach: correlate existing noise at particular frequencies

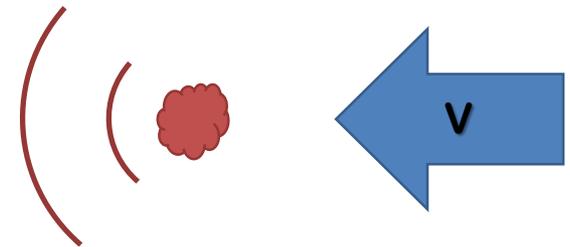
Velocity and pressure fluctuations due to flow instabilities generate acoustic noise



hydrodynamic $\tilde{p}_h \sim \rho \tilde{v}^2$

acoustic $\tilde{p}_a \sim \tilde{\rho} c^2 \sim \rho c \tilde{v}$

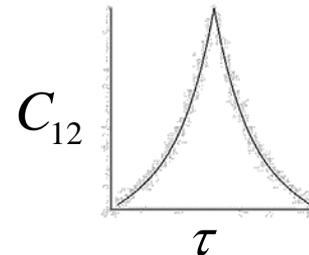
hydrodynamic energy is converted to acoustic energy



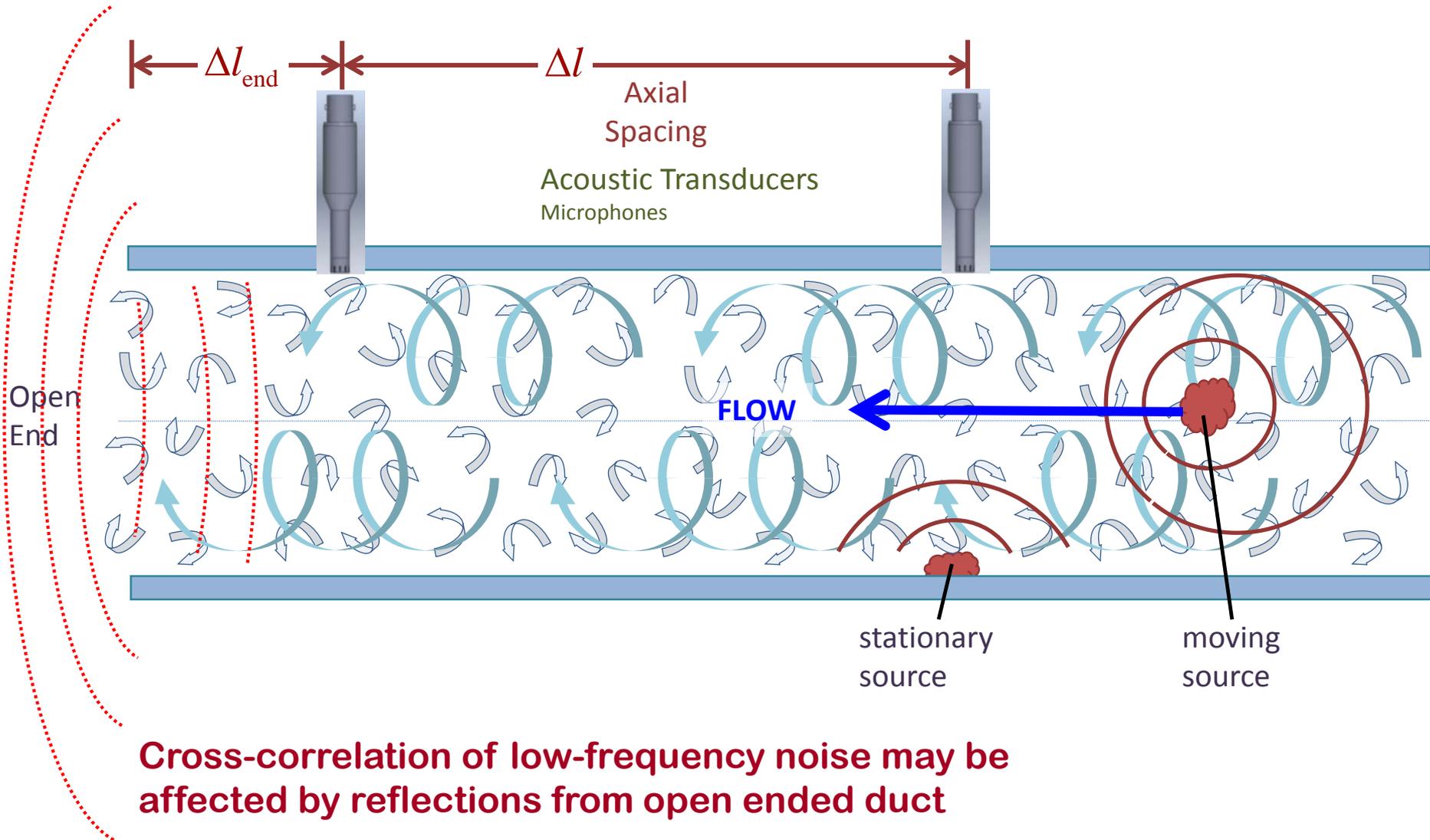
Cross-correlation of acoustic pressures

$$C_{12}(\tau) = \int_{-\infty}^{\infty} p_1(l_1, t) p_2(l_2, t + \tau) dt$$

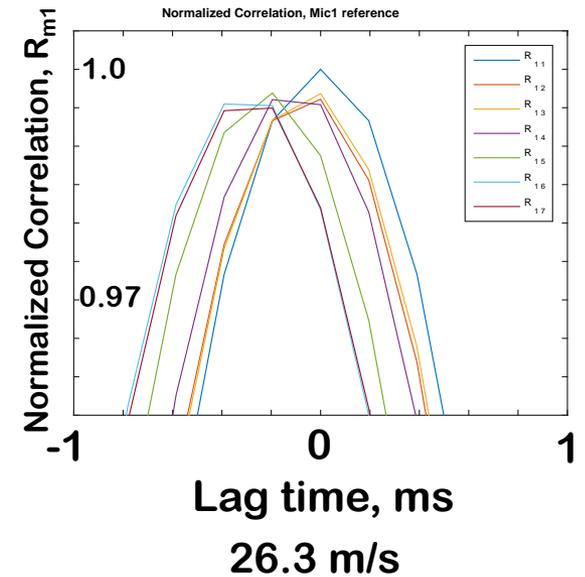
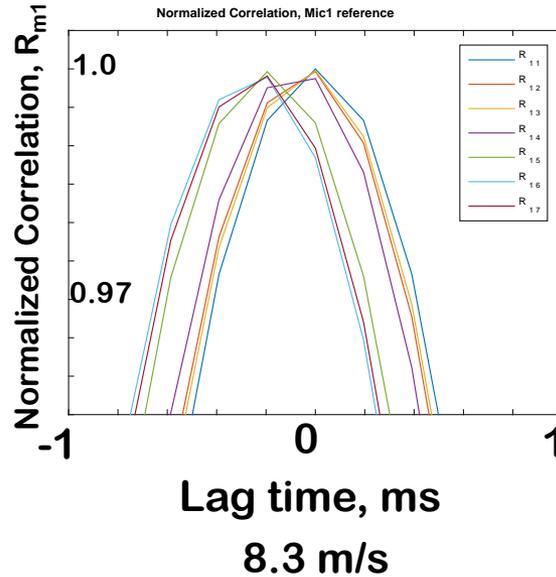
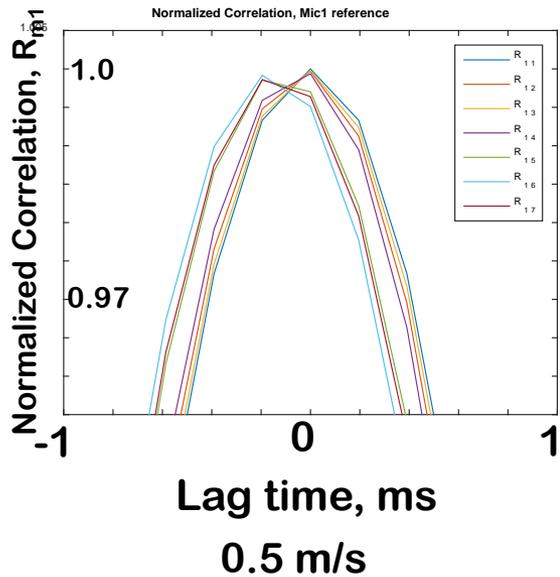
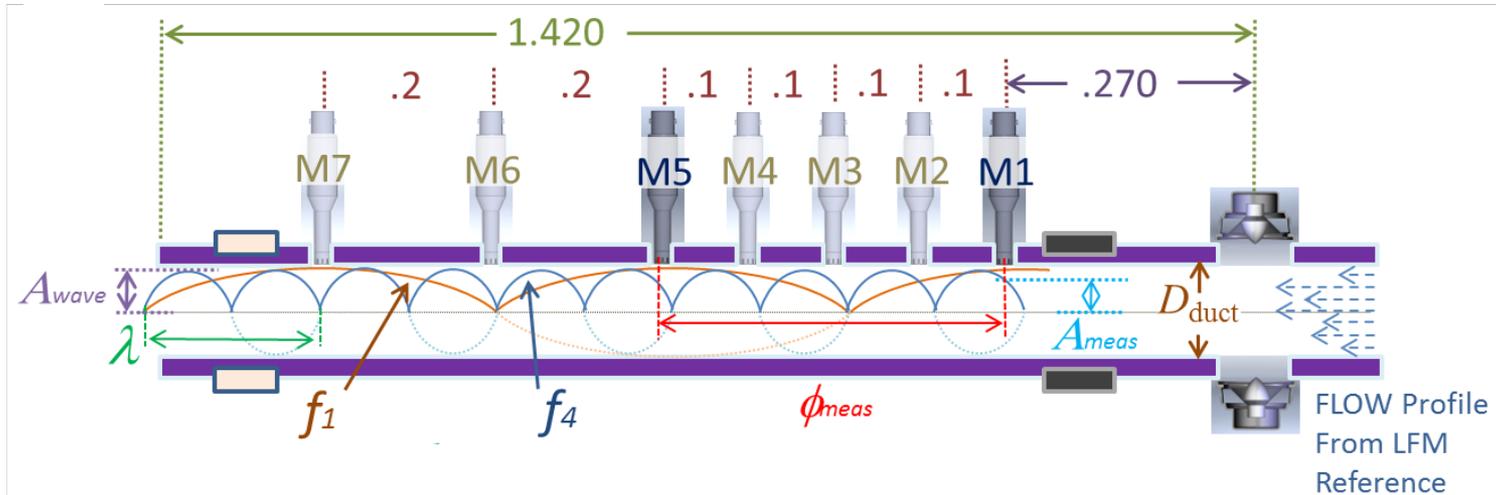
$$\max C_{12} \text{ at } \tau_{\max} \Rightarrow \frac{\Delta l}{\tau_{\max}} \approx c + V$$



Acoustic noise generated by flow confined in a duct is complicated by reflections



Correlated noise in 1:100th scale LWAF

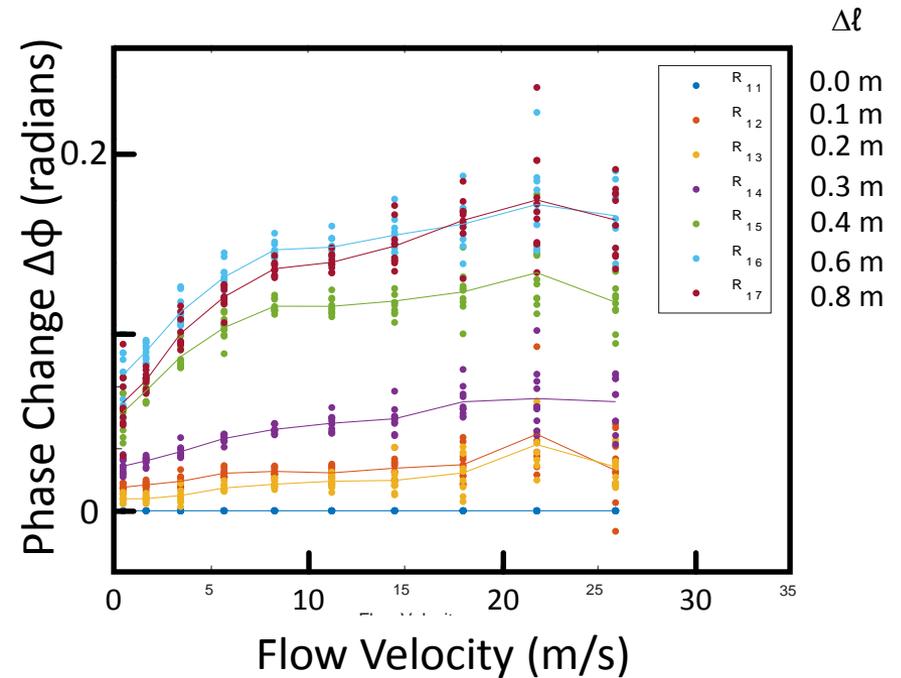
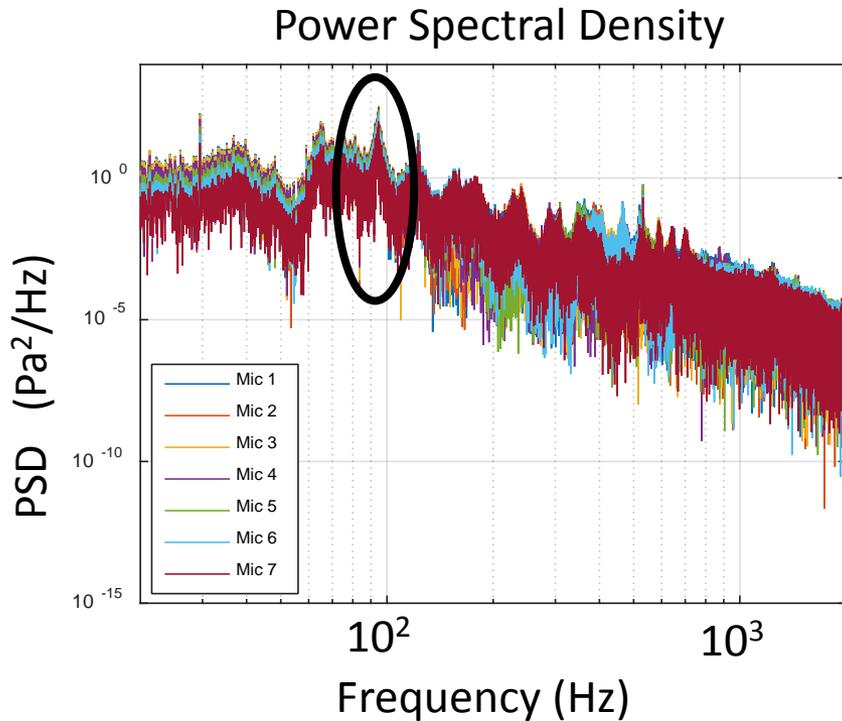


Correlated noise in 1:100th scale LWAF at 94 Hz

LWAF diameter ≈ 0.10 m

Plane wave sound for frequency < 2000 Hz

Preliminary measurements



Filtered Correlations,
94 Hz Center Frequency, 1 Hz bandwidth

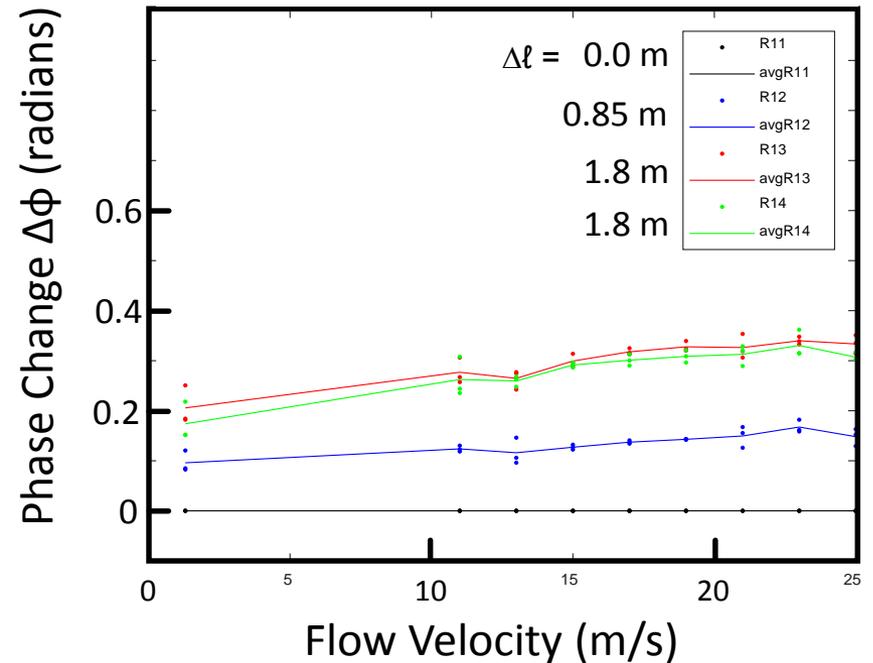
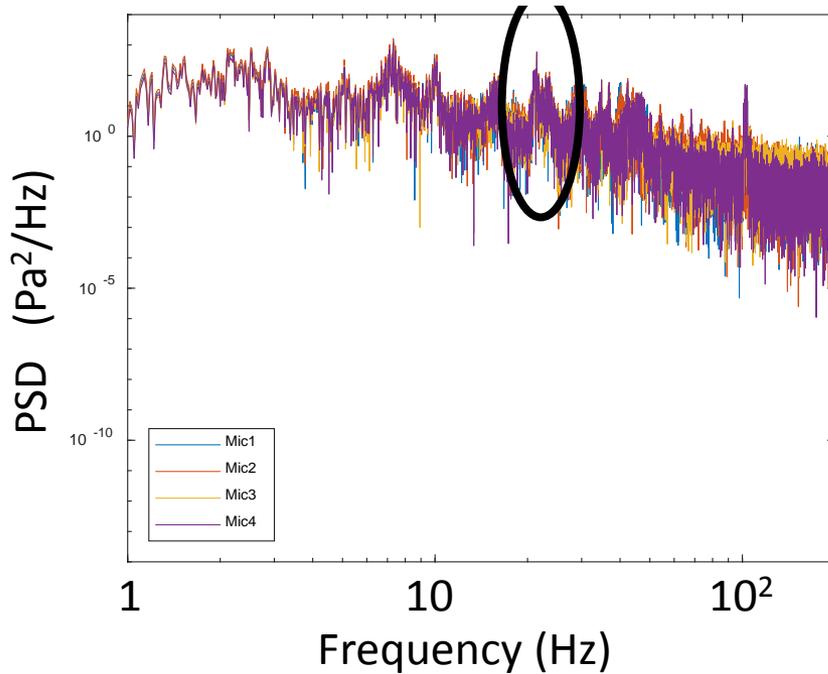
Correlations in 1:10th Scale Model Smokestack Simulator

SMSS diameter ≈ 1.2 m

Plane wave sound for frequency < 170 Hz

Preliminary measurements

Power Spectral Density



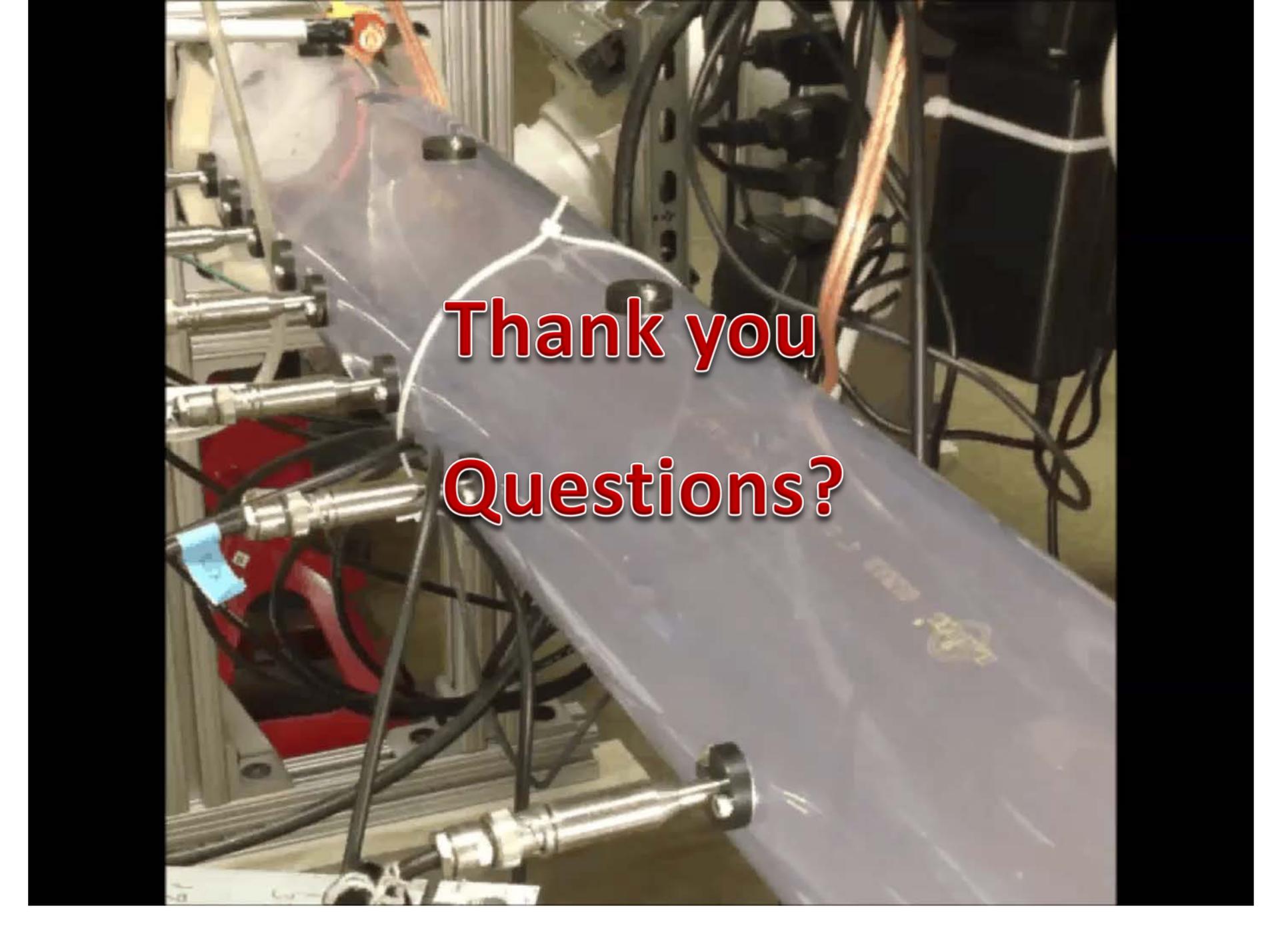
Filtered Correlations,
21 Hz Center Frequency, 1 Hz bandwidth

Summary and Future Work

- **Active Sound Generation:**
 - Success measuring flow to $u(V) \leq 1\%$ for 1:100th scale test model
- **Passive Sound Generation:**
 - 1:100th scale
 - Acoustic model predicts phase change as a function of velocity and frequency
 - Qualitative agreement with model
 - 1:10th scale
 - Measured phase changes with respect to flow velocity

Summary and Future Work

- **Develop acoustic model for 1:10th SMSS**
 - **Theoretical model for correlations**
 - **Determine relation between flow velocity and measured phase change**
 - **Demonstrate measurement of V**
 - **Determine the uncertainty $u(V)$**
 - **Goal: $u(V) \leq 1\%$ relative to SMSS calibrated reference section**



**Thank you
Questions?**